

Propylene Loop Heat Pipe Development Model Charles Baker/Orbital Michael Nikitkin/DCI Jentung Ku/GSFC Dan Butler/GSFC Tarik Kaya/ISU

March 1, 2000 Spacecraft Thermal Control Technology Workshop



Flight LHP System (Laser)

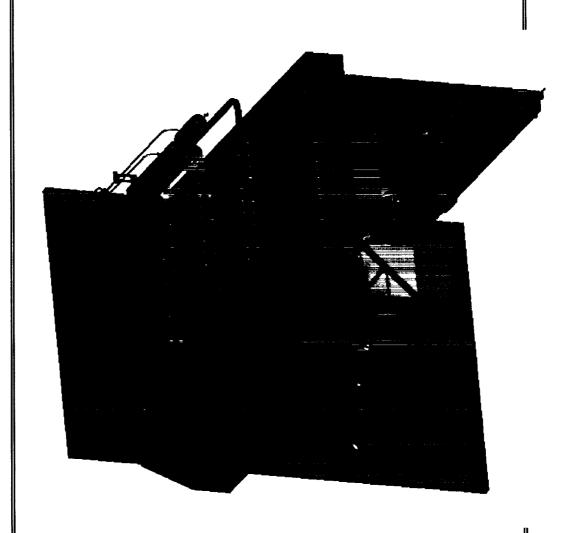


•Three HPs linked to Propylene LHP

Radiator Panel Loop Heat Pipe •LHP Condenser embedded into two-sided radiator panel Heat Pipes Lasers

Flight LHP System (Laser)







Test Design and Objectives

•Test Design

- spacers, minimize the Temperature Difference by Heating Stand) Conductively isolate all components from the Test Stand (G10
- Radiatively isolate all components from each other with MLI
- •Use heater plates to control sink for both sides of radiator
- •Test Objectives for Both Hot and Cold Cases
- Demonstrate LHP system startup and operation with propylene
- Demonstrate LHP Temperature Control
- Measure Control Heater Power with On/Off Controller and Liquid-Vapor Line Coupling blocks
- Compare Adverse to Reflux Orientation Operation



DM LHP Test Design

Two Mass Simulators linked

to Two Al-NH3 HPs

Single Mass Simulator (15

Kg) is actively powered to

resemble 1 laser

e I laser

•Double Mass Simulator (30 Kg) resembles 2 un-

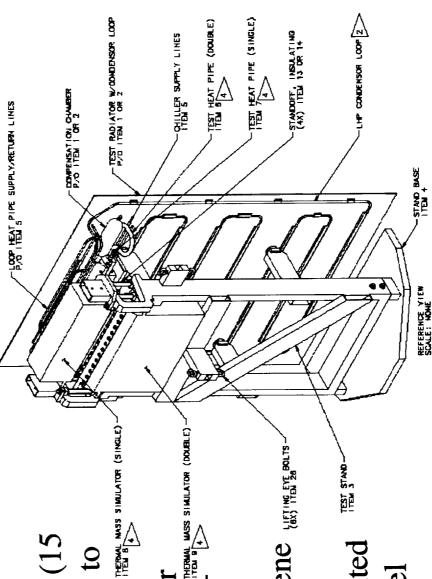
Two HPs linked to Propylene 個別報告

powered lasers

LHP

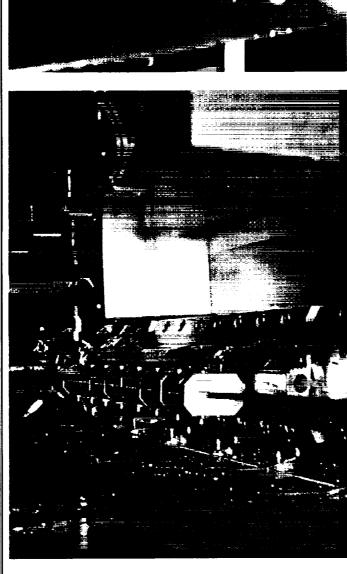
•LHP Condenser flange bolted

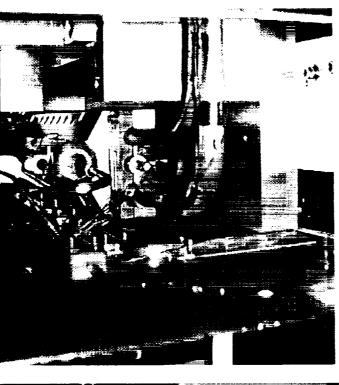
into two-sided radiator panel





Starter Heater and Coupling Blocks





Ten Al Blocks (1" long) couple Liquid and Vapor Lines

Starter Htr is a Dale NHG-25 50 Ohm resistor footprint 0.56" x 1.1"

Heater was located 1" from end of evaporator (away from CC)

Startups were performed with 0, 15 and 20 W on heater



CC Control Heaters and PRT

 Heaters were circumferentially wrapped around the middle of the CC

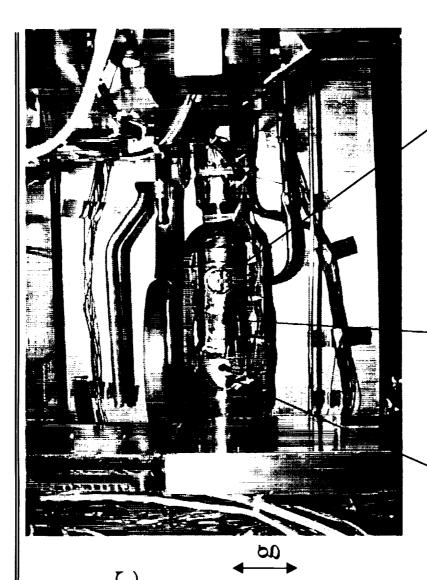
•PRT was used for control

and placed along the centerline of the CC

•On/Off Controller setpoint ↓^g

was +/-0.1°C

•Survival Thermostat had a setpoint from 0 to 5°C



Thermostat

PRT

Circumferential Control Heaters



radiator panel on one side and the -170°C TV shroud on the other side were used to Two Heater Plates which viewed the

•Plates were temperature controlled with large Kapton Heaters and painted black simulate the Flight Environment

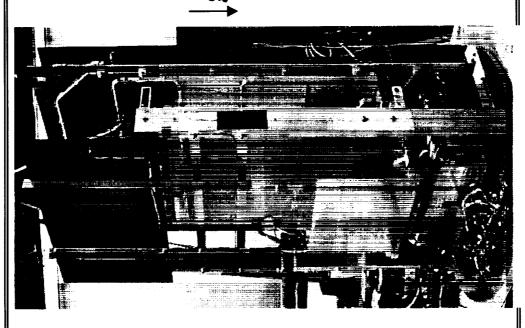
Plate Setpoints were correlated with Flight

predicts

the majority of the vertical condenser above Reflux mode (as shown) was defined as

Adverse height could be as high as 44"

the evaporator (adverse is the opposite)





Startup Tests

•17 Startups were performed with the following parameters:

Reflux versus Adverse Orientation (+44" vs -44")

•0 W, 15 W, 20 W of Starter Heater Power

•Hot and Cold Survival Sink Conditions (-100 to -45°C Teff)

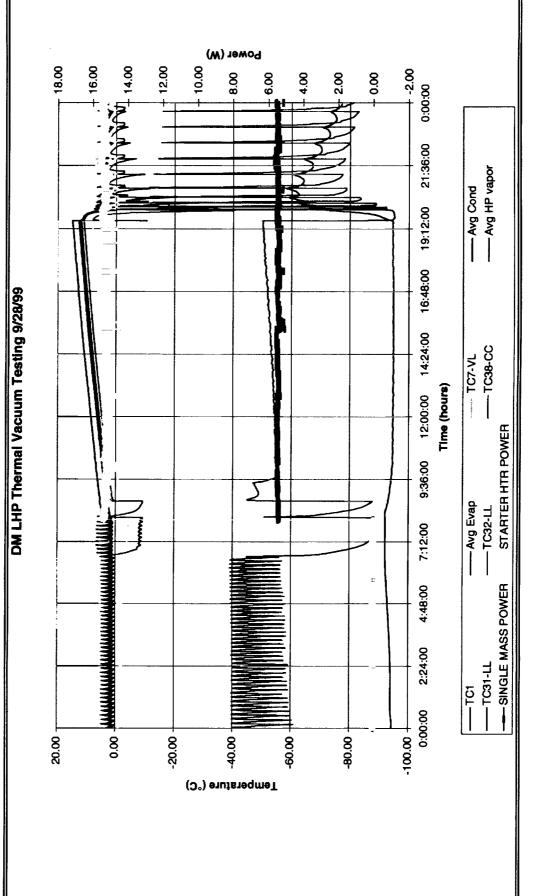
Various initial Evaporator and CC temperatures (0 to 25°C)

•All startups were preceded by pre-heating the CC 3-5°C above the evaporator temperature

Verified need for starter heater for startup



Typical Cold Startup





Startups: Effect of Orientation and

Sink

- Orientation (Adverse versus Reflux)
- •6 Startups performed in Reflux
- •11 Startups performed in Adverse
- Startups were similar in both orientations for
- Time for Startup
- Superheat at Startup (Max Evap Temp CC Temp)
- Maximum Evaporator Temperature
- Sink (Hot and Cold)
- •No significant difference was seen between the Hot and Cold Sinks
- Startup to Startup, the above parameters varied greatly



Startups: Effect of Starter Heater

Power

- Starter Heater Power
- No Starter Heater Power
- start without a starter heater (test stopped when 30°C reached) •Even with 100 W on the Mass Simulator the LHP would not
- •15 W vs 20 W of Starter Heater Power
- •The LHP started at similar evaporator temperature for either starter heater power on average (16.5°C)
- 20 W of Starter Heater Power on average (4.2 vs 3.5°C for 15 W) •The LHP required slightly higher superheat prior to startup for
 - •The LHP required a longer startup time for 15 W of Starter Heater Power on average (18.5 vs 13.5 hours for 20 W)



- Effect of Initial Evap Temp
- •All startups require the elapse of time and the evaporator reaching a high enough temperature
- •A high initial evaporator temperature still required time prior to startup, but less overall time than a cold initial evaporator temperature
- •The LHP always started before reaching 20°C as long as the evaporator was below 15°C initially



Startups: Repeatability

Repeatability

•5 Similar Tests were conducted with 15 W on Starter Heater

•12 other tests had a variety of conditions (varied initial evaporator temperatures)

Startup Tests	L	Avg of	Std Dev of	Avg of Max	Std Dev of	Time for	Std Dev of	Avg	Std Dev
	Tests	Max TC 1	Max TC 1	Overall Evap	Max Overall	Startup	Time for	Superheat	
		Temps	Temps	Temps	Evap Temps	(hours)	Startup		
All with	17	18.1°C	3.9°E	15.7°C	3.7°C	12:20	7:07	4.0°C	0.7°C
Starter									
Heater									
(T0<20°C)									
20 W Starter	12	17.7°C	43°C	15.1°C	4.3°C	8:30	5:06	4.2°C	0.7°C
Heater Only									
15 W Starter	5	18.9°C	2.5°C	16.6°C	2.5°C	18:28	5:31	3.5°C	0.4°C
Heater Only									



CC Control Heater Power Tests for CC Temperature Control

- Control Heater power can only be accurately measured in TV
- •GLAS DM LHP may be the first control heater power measurements in TV
- •20 Control Heater Power Test measurements were performed with the following parameters:
- •Reflux versus Adverse Orientation
- Hot and Cold Survival Sink Conditions and control setpoint
- •Mass Simulator Power (100 W, 120 W, 200 W)
- Liquid-vapor Coupling blocks (8 vs 10 blocks)
- Changing control setpoint (increasing and decreasing)
- Measured temperature stability at mass simulator
- All control heater power measurements were verified electronically and with a strip chart recorder



Heater Power: Effect of Orientation

	TC32	1:	-1.	5.
rientation	ТСЗІ	-51.	-18.	-22.
Adverse Orientation	Power Required (W)	2.7	1.5	2.1
	Test#	14	11	81
	TC32	1.	-3.	3.
Reflux Orientation	TC31	-99	-20.	-27.
	Power Required (W)	2.6	1.9	2.4
	Test#	6	12	13

•Three comparable tests were performed in both the reflux and adverse

between the control power requirements between the orientations Cold case Test 9 and Test 14 show no significant differences

Hot Case Tests 12 and 13 and comparably 17 and 18 respectively

warmer liquid return temperatures (probably a warmer sink) as Control power differences can entirely be explained by the seen in TC32

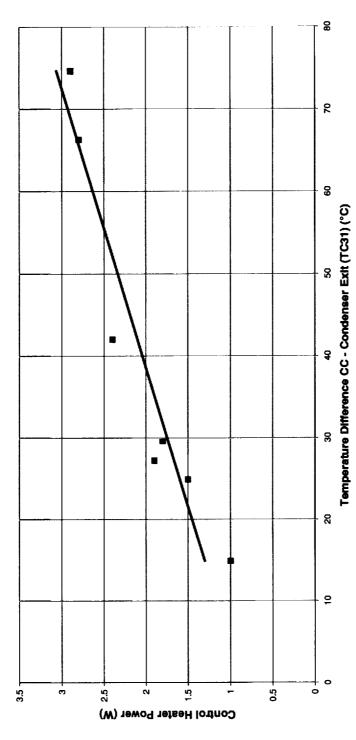
•Note: TC31 is at the exit of the condenser, TC32 is after the liq/vapor •No significant differences in heater power are observed when comparing two orientations in the hot and cold sinks tested

coupling blocks



Heater Power: Effect of Sink and Heater Setpoint





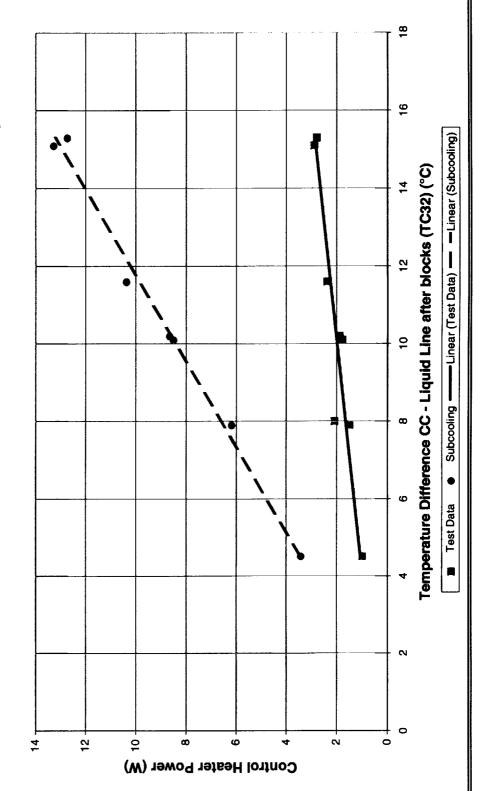
•CC setpoint were varied between 6.5 and 14.5 °C

 The greater the difference between the CC setpoint and the condenser exit temperature, the greater the control heater power requirement

Geoscience Mitimeter System

Heater Power: Comparison With Subcooling

DM LHP Thermal Vacuum Testing Overall Control Heater Power for all 120 W Tests Only





Heater Power: Effect of # coupling

blocks

 Coupling blocks had a fairly uniform coupling under orientations (Test 1 had 8 blocks, Test 3 to 20 had 10 a wide variety of input powers, sink conditions and blocks)

•Coupling blocks may be modelled as a fixed-	conductance using a log mean temperature difference
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- •Variations in results cannot be explained with
 - variations in test conditions
- Coupling was dominated by laminar film coefficient on liquid line
- •Even with the enhanced coupling vs. laminar, the heater power requirement was ~1/3 of model predictions

C per block (W/K)	620'0	0.081	0.092	260'0	880.0	0.070	060'0	0.084	0.077	880'0	0.081	660'0	101.0	0.072	0.086	0.010	0.113	
Test #	1	3	6	10	11	12	13	14	15	16	17	18	19	20	avg	std dev	std	dev/avg



Heater Power: Effect of Evap

Power

evaporator powers: 100 W, 120 W, 201 W; the control heater power Control Heater Power was compared for a wide variety of applied was independent of evaporator power in this range

•This can only be explained through assuming that the CC is isolated from the liquid core and partly coupled to liquid return line •The CC is coupled to the liquid return through a fixed conductance (0.19 W/K-6" of coupling) as measured in heater power

Liquid return line had steady state Reynolds numbers below 2000

•The heat leak through the wick is based primarily on the liquid

return line temperature, not pressure losses in system

 Mass flow rate must be adjusted for difference between sensible heat of liquid returned and actual control heater power based on

fixed coupling



Changing Control Heater Power:

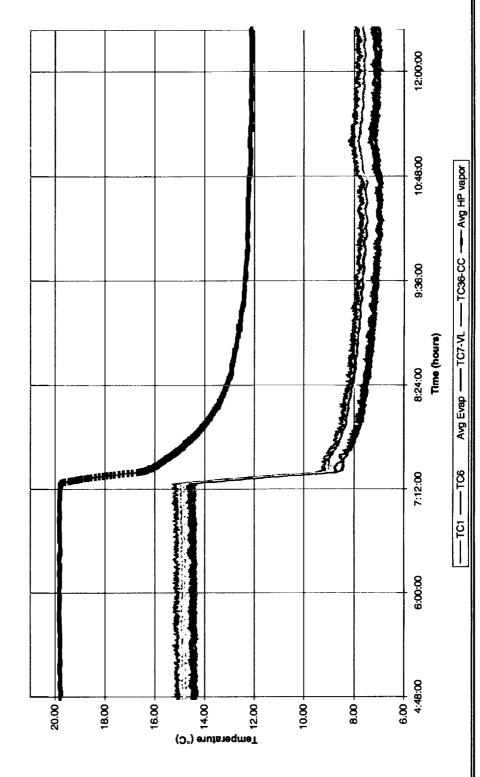
Setpoint

- •Raising the control setpoint results in diverting heat dissipation to sensibly heating the mass simulators
- The fastest setpoint rise possible was 0.6°C/5 minutes without shutting down the LHP
- •If the starter heater is activated, the setpoint may be raised faster •No limitations were seen in the rate at which the setpoint is
- decreased 14°C at a step was attempted (LHP Un-controlled dropped 14°C in 30 minutes)
- Stabilization times are on the order of hours



Control Temperature Stability







Conclusions

- A starter heater is required for startup of large thermal mass propylene LHP systems (also verified by independent testing at JPL and Dynatherm)
- Startup occurs after hours of time pass and the evaporator temperature rises to or above a "turn-over temperature" (which results in the Delta-T or "superheat" required between Evaporator and CC)
- Propylene LHP control heater power is solely a function of the difference between the liquid return line temperature and the CC setpoint times a constant conductance coupling
- Utilizing liquid-vapor coupling blocks is a highly effective way to decrease control power without compromising control in the hot case
- •Test measured 1/3 of predicted control power requirement
- •Temperature control of a mass simulator is possible to +/-0.1°C